



Integrated Energy Systems at the University of Maryland

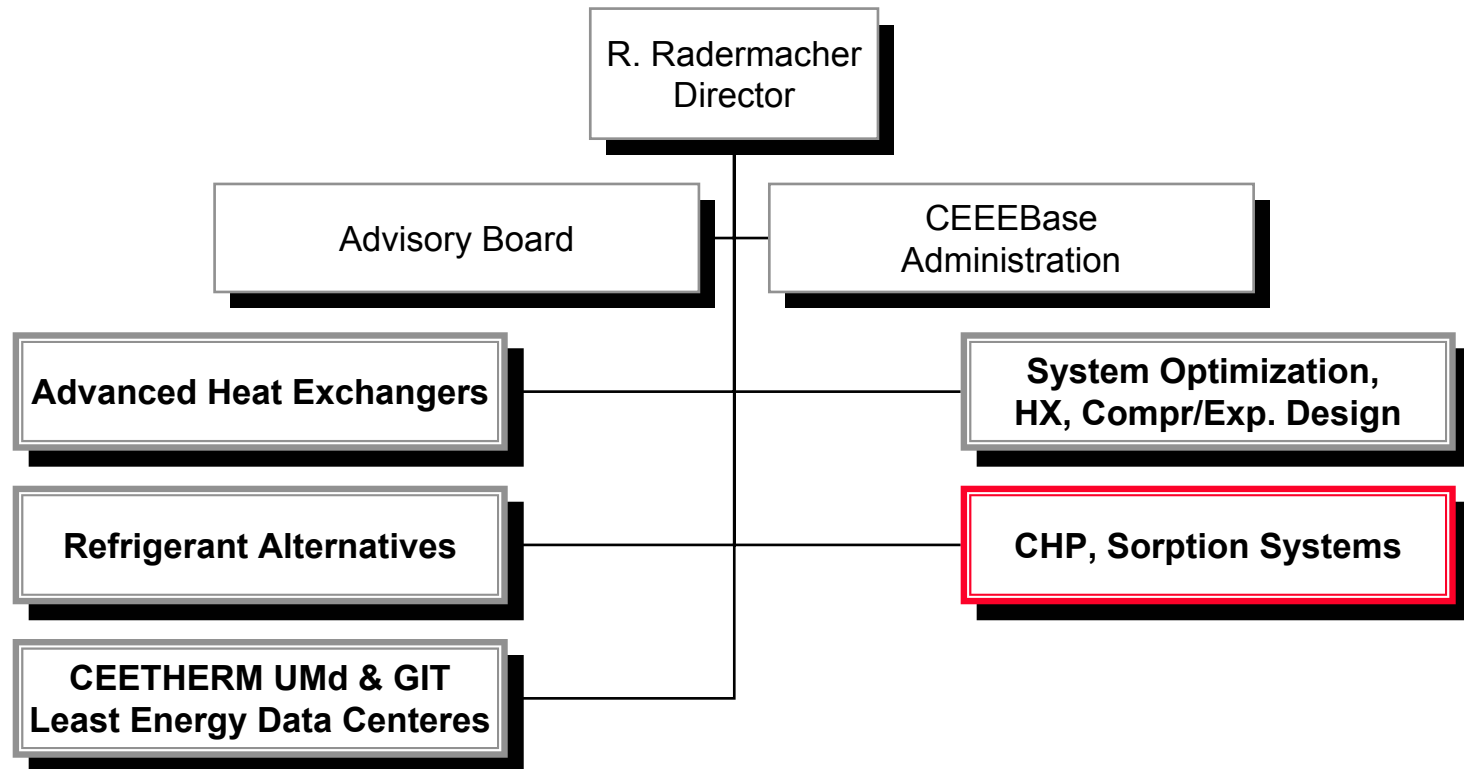
April, 2002

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Reinhard Radermacher



CEEE Organization



Objective

- Investigate Integration Issues of CHP Components into Systems
- Demonstrate Performance Potential
- Address Research Needs as they are Presented by Consortium Sponsors (not DOE funded, but grew out of this project)
 - Modeling and Optimization
 - Diagnosticians

Integration Test Center Building

- **TYPICAL, MEDIUM SIZE OFFICE BUILDING, 51,000 FT²,**
- **4 FLOORS, 2 ZONES**
- **90-TON ROOF TOP UNIT FOR EACH ZONE**
- **COOLING ALL YEAR**
- **ECONOMIZER CYCLE**
- **VAV WITH ELECTRIC REHEAT**
- **LOW LEVEL CONTROLS**
- **GAS RARELY USED**
- **ELECTRIC ~300 kW PEAK**
- **10 YEARS OLD, 200 OCCUPANTS**





Start

Time of Last Reading
Wed 28/Feb/2001 1:00:37 PM

Microturbine Efficiency (%)
24.21

* Estimate (as of 2/27/01)

Damp1
0: 4.446m

Damp2
0: 3.894m

Delay
0

Interval
60

CO2 (ppm)

R1 S.A. 0: 410.9

R1 R.A. 0: 480.4

R2 S.A. 0: 489.5

R2 R.A. 0: 529

Total Accumulated Actual Savings (kW-hr) *
-237.84

Total Accumulated Potential Savings (kW-hr) *
1285.05

Total Accumulated Actual Savings (\$) *
\$ -3.44

Total Accumulated Potential Savings (\$) *
\$ 25.88

..\\DATA\\HP VEE\\022811.dat

Temperature (F)

R1 SA 50.99

R1 RA 71.11

R1 MA -13

Space... 00: 74.92

R2 SA 59.11

R2 RA 72.19

R2 MA 54.93

01: 73.99

02: 76.51

Relative Humidity (RH)

R1 S.A. 19.46

R1 R.A. 12.93

R1 M.A. 264.1

Space... 00: 12.69

R2 S.A. 16.75

R2 R.A. 12.29

R2 M.A. 27.18

01: 14.26

02: 13.97

Gas line

Flow Rate (CFM) 0: 566.6E36

Comp. ... 2.379

Temp (C) 0: -1.228E39

Temp(deg. F) 0: 2.944E39

RH (%) 0: 122.6

P (psia) 0: 495E36

Pressure (psig) 0: 113.5E36

\\SRI-DATA\\SRI-022811.dat

\\RI-DATA\\SRI-Press022811.1

SRI Testing

4-Hour Space Temperature History (F)

4-Hour RTU Temp History (F)

4-Hour Space RH History (%)

4-Hour RTU RH History (%)

4-Hour Power History (kW)

4-Hour Flow Rate History (CFM)

Power (kW)

T1 15

HP1&2 27.8

RTU1 9.26

CFM 0: 2866

25.28

46.7

15.56

1: 2324

T2 14

HP3&4 23.2

RTU2 10.7

2: 8634

23.5

39.03

18.04

3: 9525

Total 168.1

kW 0: 103.8

V 0: 1.04

4: 12.98k

5: 10.25k

RTU1 RTU2

1 EF

2 C1

3 C2

4 HG1 46.5 84.8

5 HG2 48.9 41.9

6 L1

7 L2

8 SF1 5.4 8.8

9 SF2 4.6 7.2

10

11

Natural Gas

ft.^3 /int. 0: 15

Bldg Press

Z1... -0.067

Z2... 0.019

2: 0

Weather Station

OA F 100

OA RH % 31

CO2... -500

Solar W/m^2

Sunny 709

Wind Speed mph 39.99

Wind Direction deg 314

Press... 29.82

Ready

Cooling Heating and Power

CHP Home	About CHP	Consortium	Technology	Tech Transfer	<input type="text"/>
CEEE Home	CHP Team	Directions	Links	Publications	

Search

[Integrated Energy Systems](#)



Discover the world of Integrated Energy Systems.

[CHP Virtual Tour](#)



Take a virtual tour of the Integration Test Center at the University of Maryland's Chesapeake Building !

[Data Acquisition System](#)



View the current conditions at the Chesapeake Building.

[Control and Monitoring](#)



Try an interactive demonstration.
username: guest
password: user

CHP Test Center

- The **Integrated Energy Systems Test Center** is designed as a research facility that explores the intricacies of integrating advanced power generating equipment, such as microturbines and fuel cells, with waste heat activated technologies such as absorption chillers and desiccant systems.
- The **Chesapeake Building**, which currently has two CHP systems installed, is a fully instrumented platform for conducting research on advanced energy efficient and environmentally friendly building technology.



CHP System 2

What's New

- [Chinese Delegation](#) Visit the CHP Integration Test Center on March 26, 2002.
- [British Parliament Members](#) Visit the CHP Integration Test Center on January 30, 2002.
- The recent [Microturbine Applications Workshop](#) was held at the University of Maryland Inn and Conference Center on January 17-18, 2002.
- The Sem-Annual CHP Consortium meeting was held in Atlantic City on Jan. 12, 2002, from 3:00 pm to 6:00 pm.
- The CHP team welcomes the newest member !

[\[More Information...\]](#)

Data Acquisition System



View the current conditions at the Chesapeake Building.

Control and Monitoring



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Tech Briefs



Browse the DOE Tech Briefs and Program Plans for CHP.

CHP Guidebook



View the Guidebook of the CHP Test Center.

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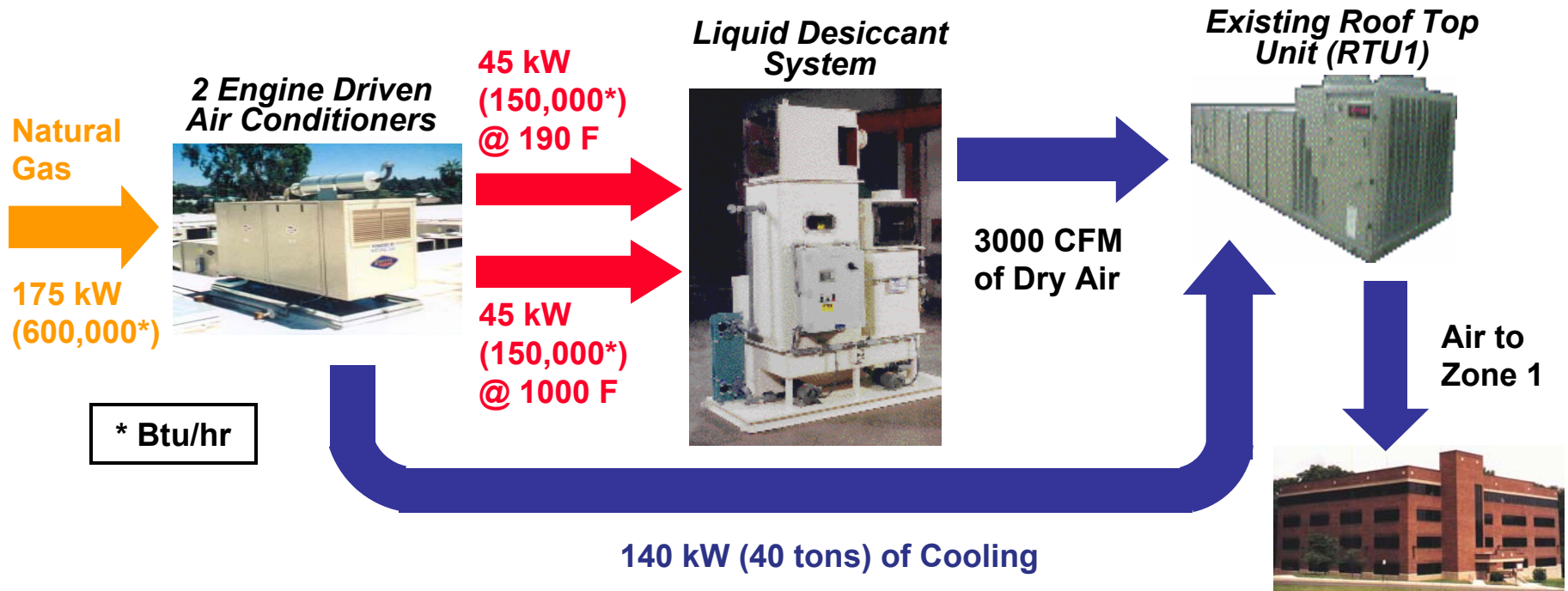
[\[More Information...\]](#)

Events

- **February 28, 2002** - International District Energy Association (IDEA) annual conference in Denver, Colorado.
- **March 12, 2002** - DG & On-site Power Conference in Atlanta, GA.
- **May 2002** - iTHERM at the Georgia Institute of Technology.
- **June 22-26, 2000** - ASHRAE Summer Meeting, Symposium on CHP for Buildings in Honolulu, HI.
- **August 2002** - IHTC in Grenoble, France.
- **September 17 - 20, 2002** - 5th IIR-Gustav Lorentzen Conference on Natural Working Fluids will be held in Guangzhou, China.
- **September 24-27, 2002** - International Sorption Heat Pump Conference will be held in Shanghai, China.
- **October 9-11, 2002** - Association of Energy Engineers (AEE) 2002 Combined Heat & Power Expo / Cogeneration Congress in Atlanta, Georgia.
- **November 17-22, 2002** - American Society of Mechanical Engineers (ASME) - International Mechanical Engineering Congress and Exposition, New Orleans, LA.

[\[More Information...\]](#)

CHP System 1



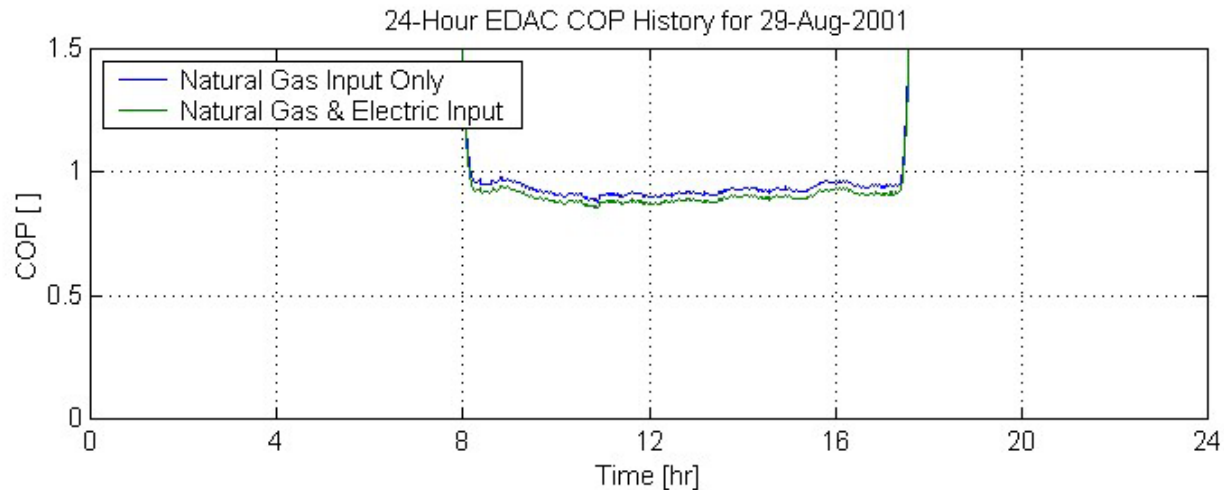
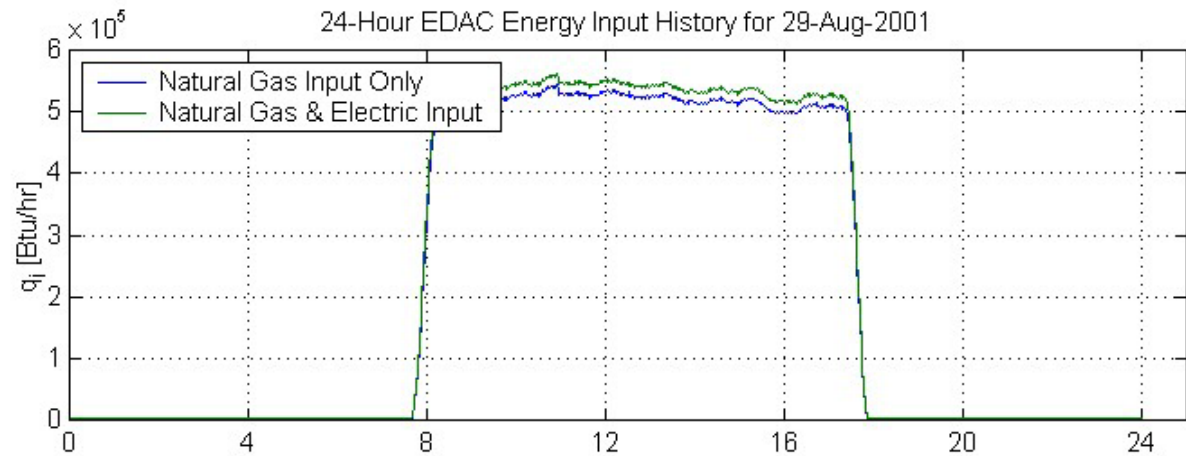
- Engine jacket water & exhaust used to regenerate desiccant

Engine Driven Heat Pumps

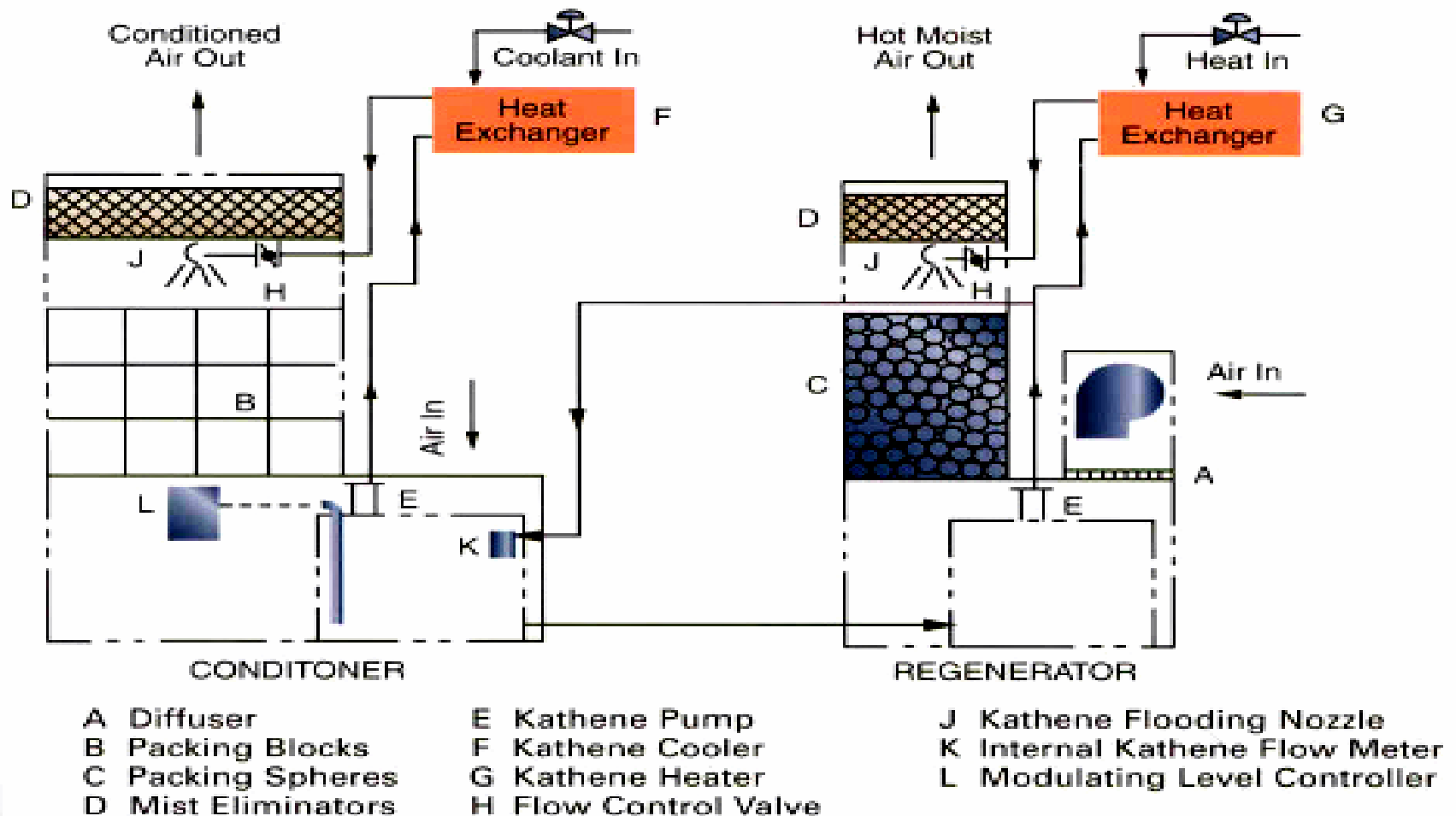
Capacity on air side, issues with air flow rate measurements

- ⇒ Power Consumption @ rated capacity, 6.8 kW measured, 8.5 kW rated
 - ⇒ 2 hp, Heat Recovery Pump, 1.5 kW
 - ⇒ 2 Condenser (outdoor unit) rated 6 kW
 - ⇒ 2 Engine section 1 kW
- ⇒ Gas consumption measured 1,050,000 Btu/hr, 10.5 therms, 1050 CFH, should be less as gas pressure is 10" wc

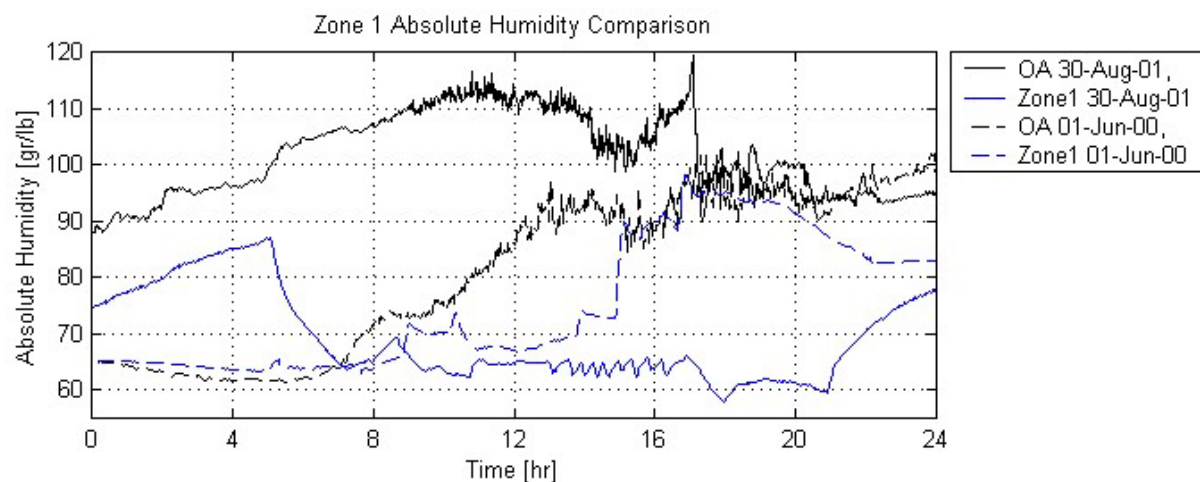
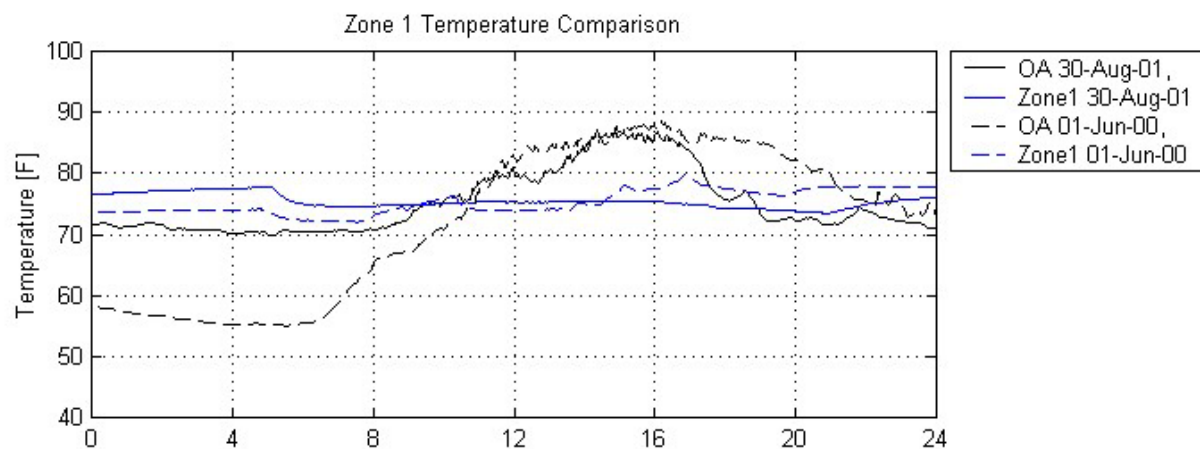
Engine Driven Heat Pumps



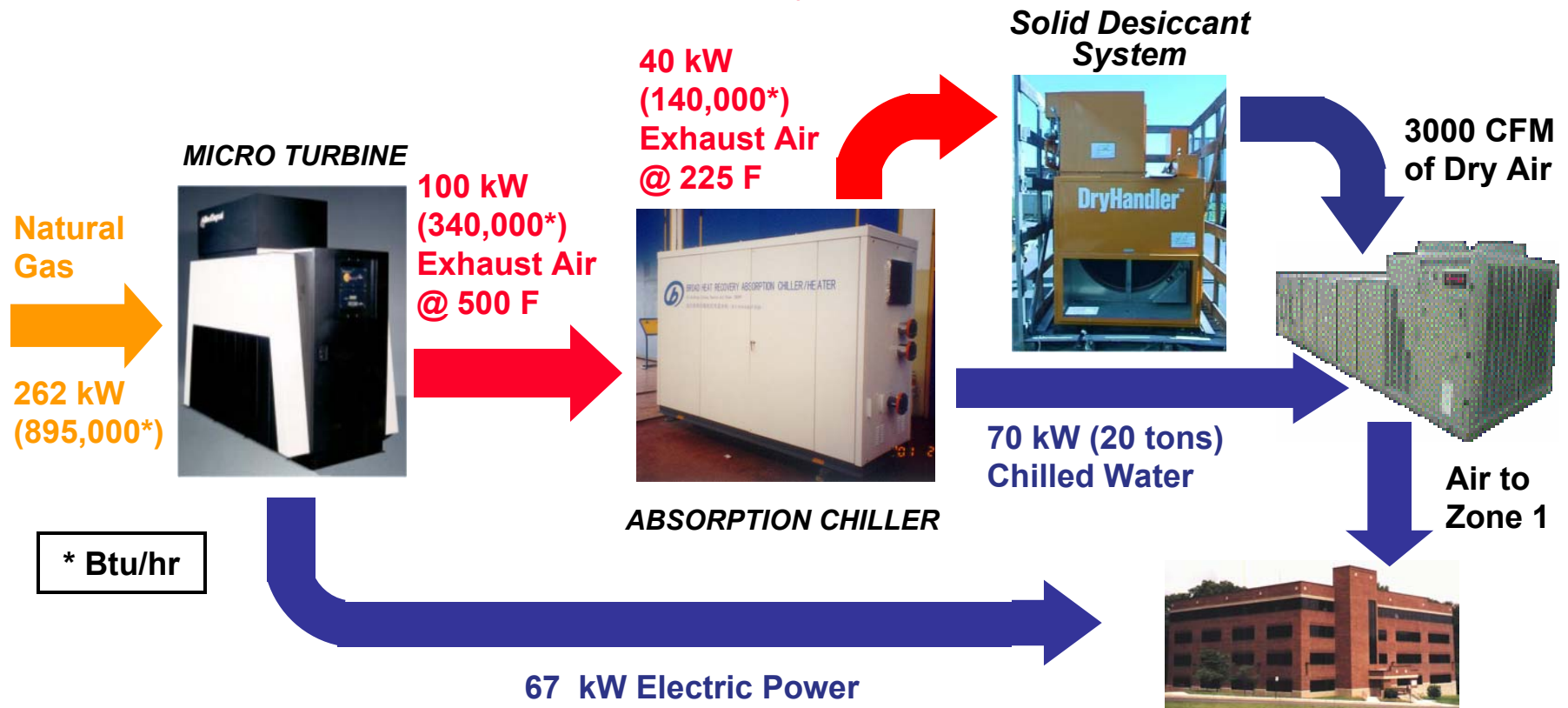
CHP System 1 Liquid Desiccant



Comparison



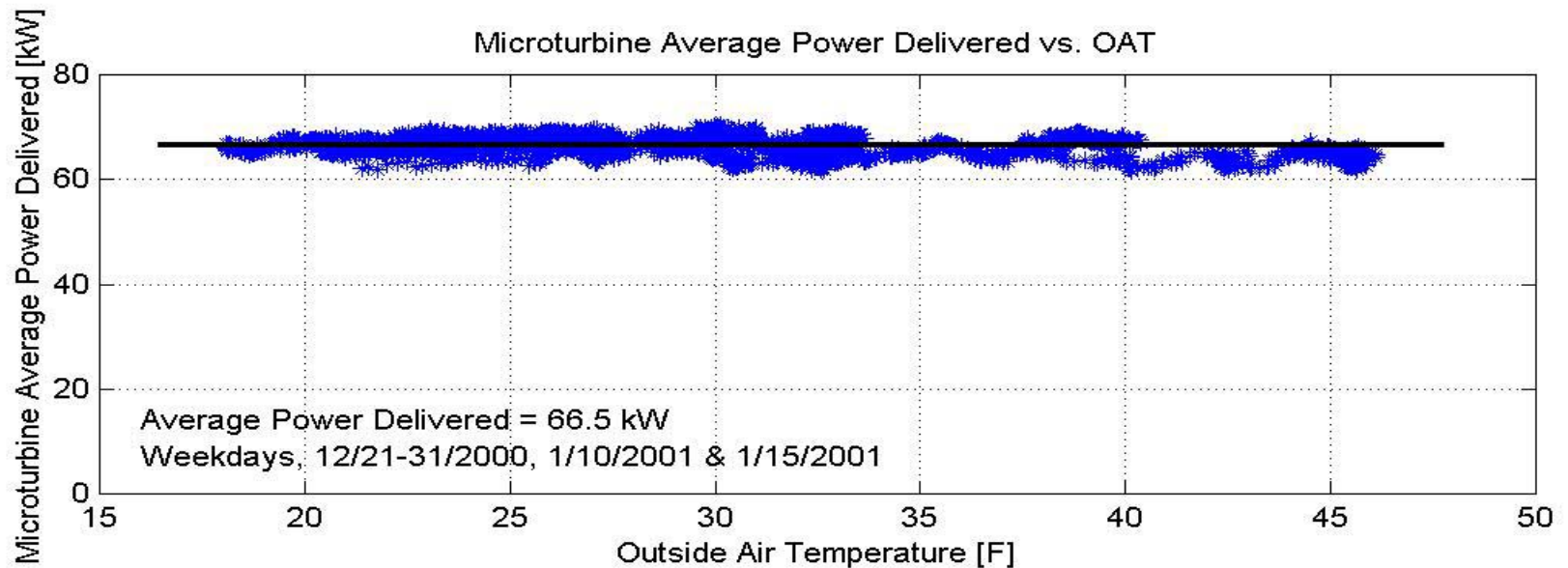
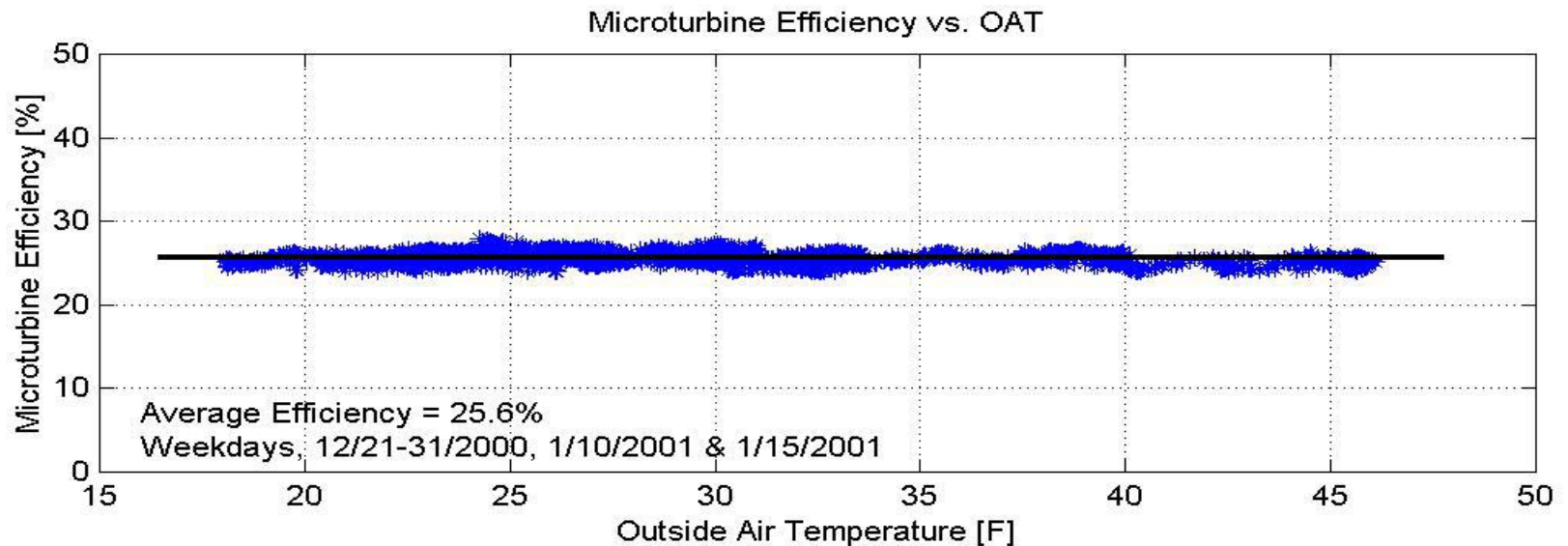
CHP System 2

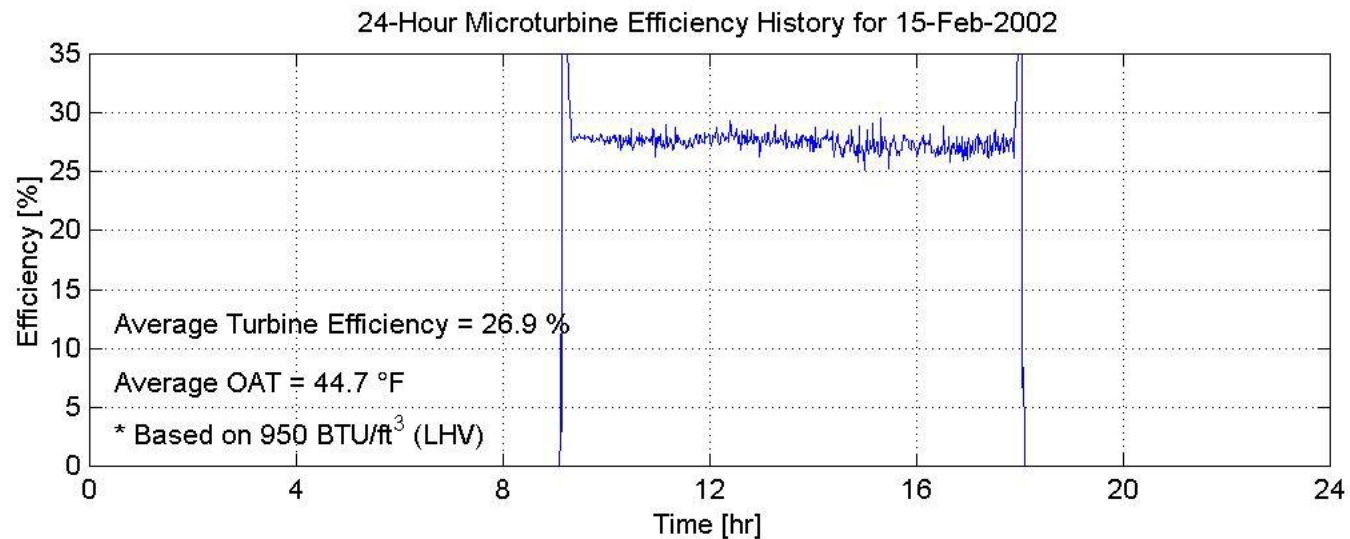
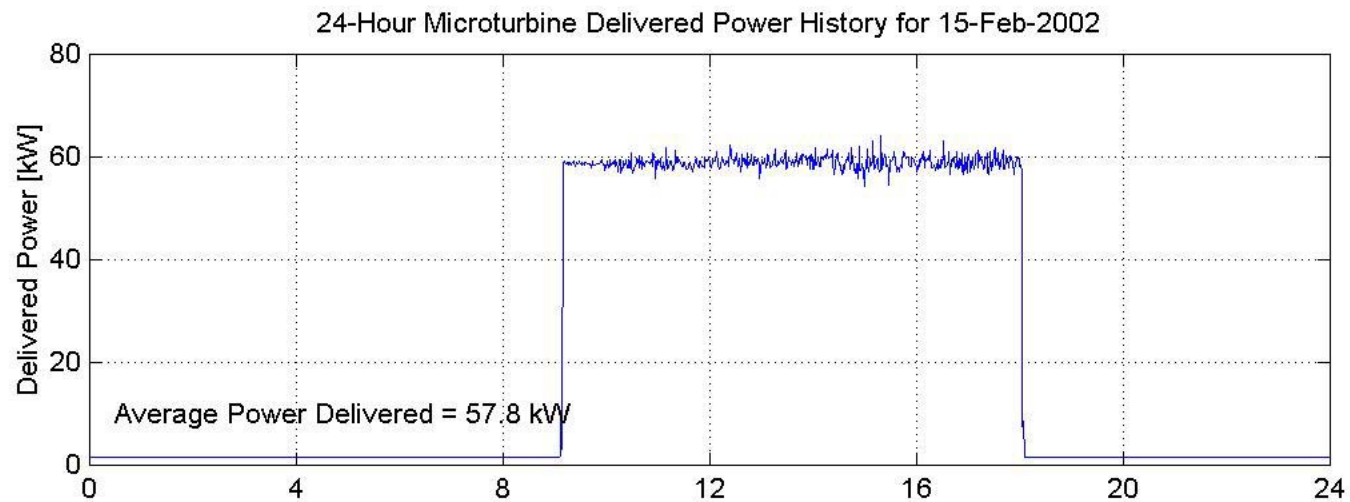


- Turbine efficiency 25.6 %, with chiller 63.5 %, and with desiccant 79.2%
- Single effect absorption chiller with COP of 0.7
- Supplemental cooling provided by existing RTU



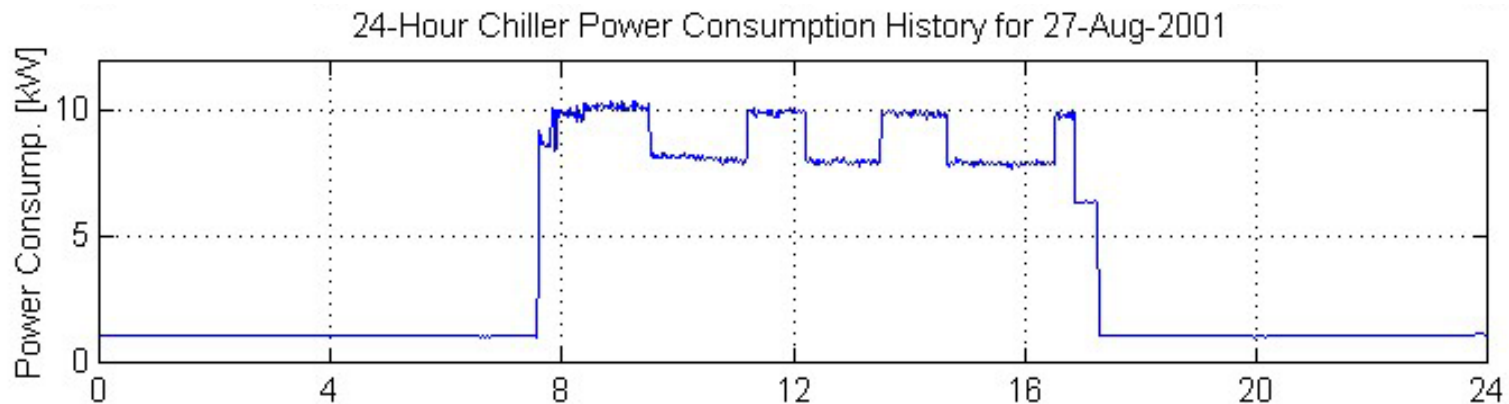
Microturbine Performance



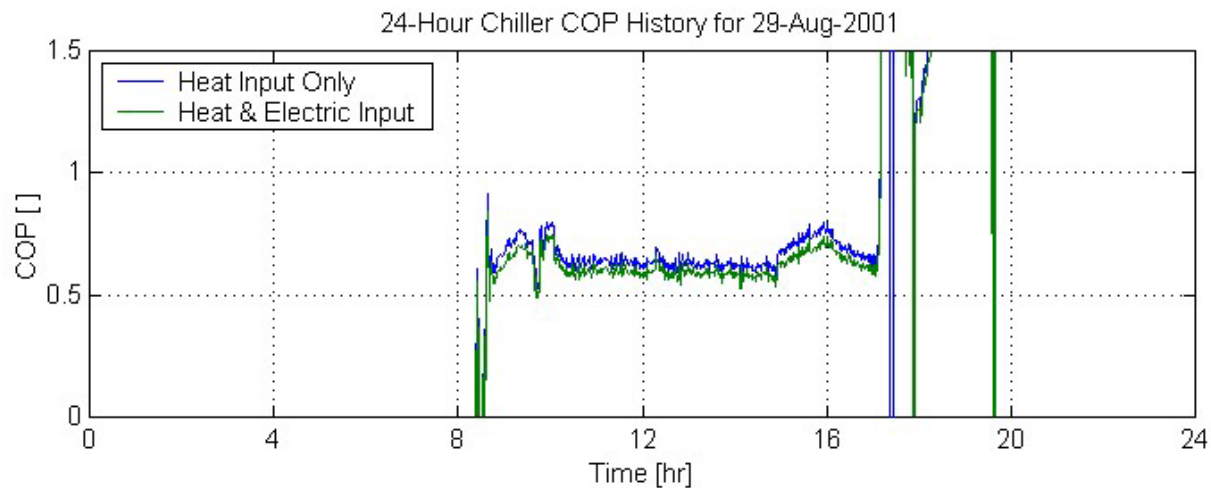
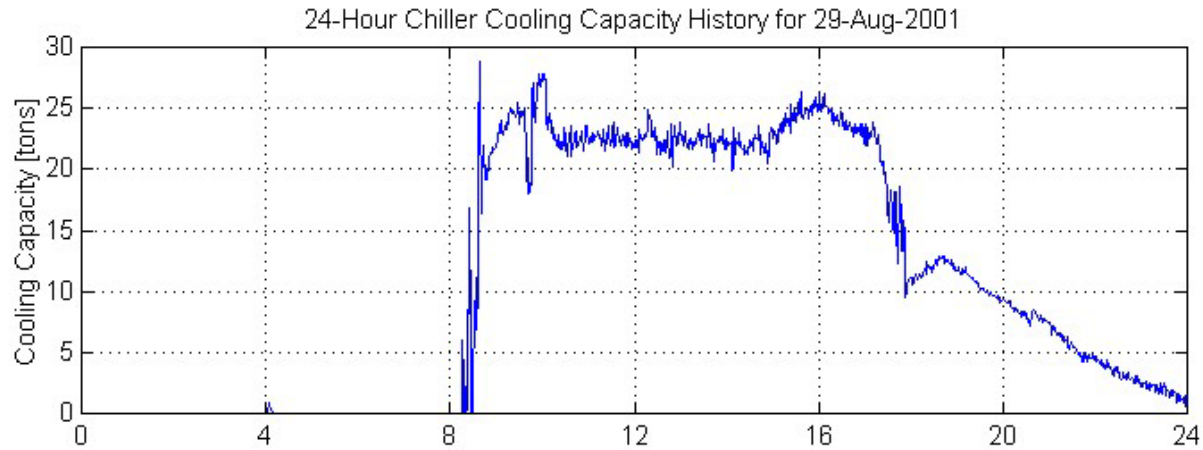


Broad Absorption Chiller

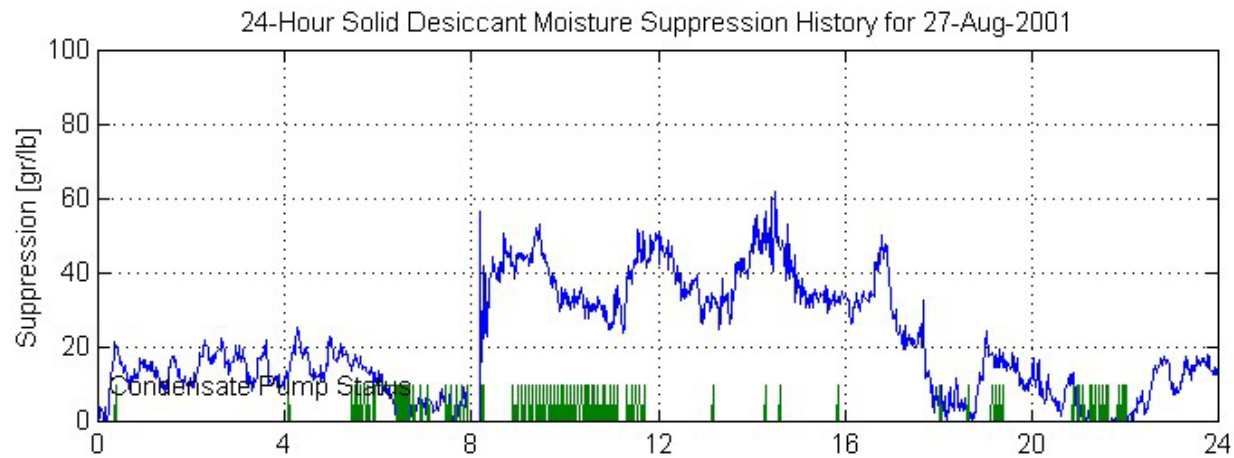
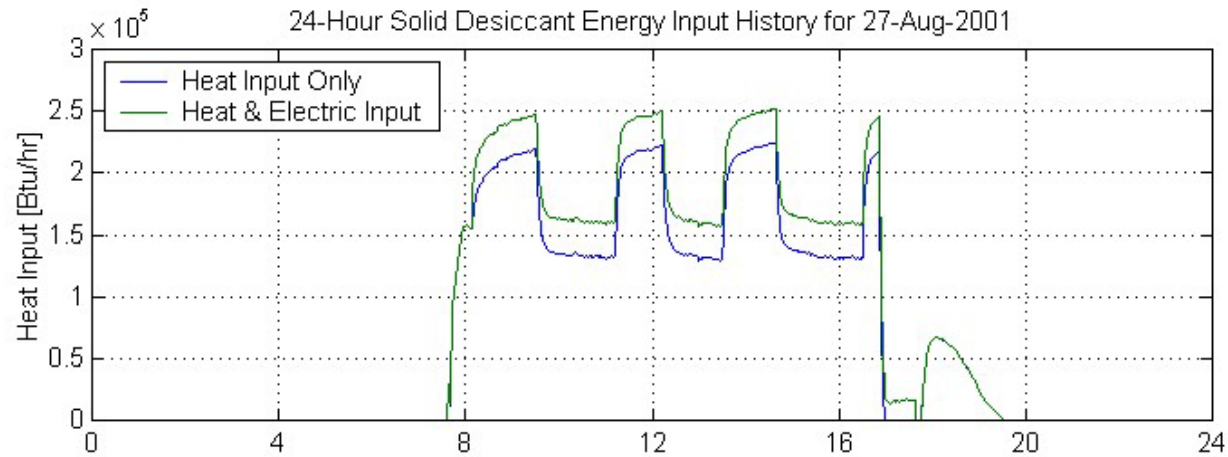
- ⇒ Microturbine exhaust driven only
- ⇒ 20-ton nominal capacity
- ⇒ Single effect, COP = 0.7
- ⇒ VFD fan drives exhaust through generator
- ⇒ Microturbine power drives chiller & auxiliaries about 10 kW
 - ⇒ Chilled water pump
 - ⇒ Condenser water pump
 - ⇒ Cooling tower
 - ⇒ Exhaust fan
 - ⇒ Solution and refrigerant pumps



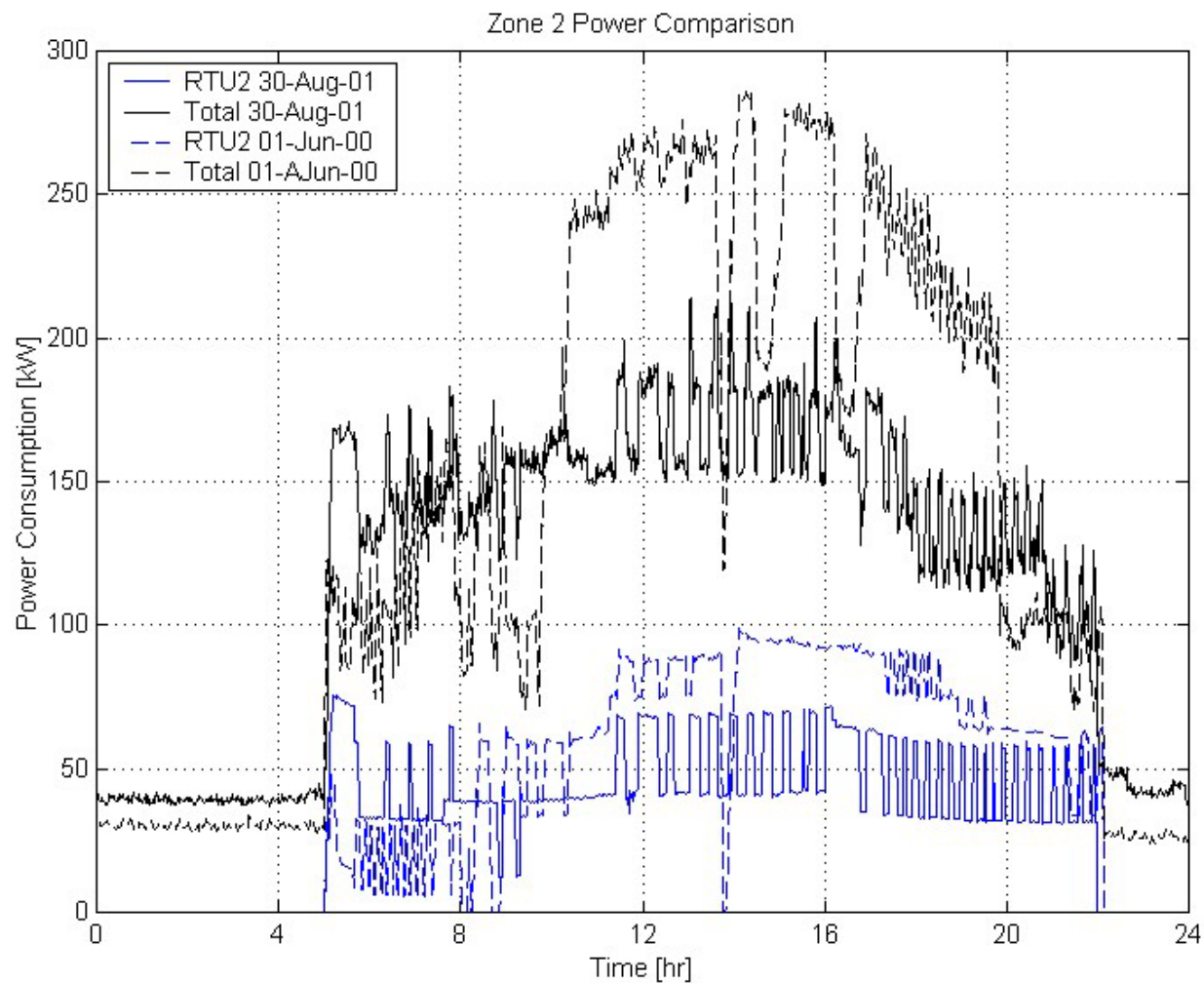
Absorption Chiller Data



Solid Desiccant Performance

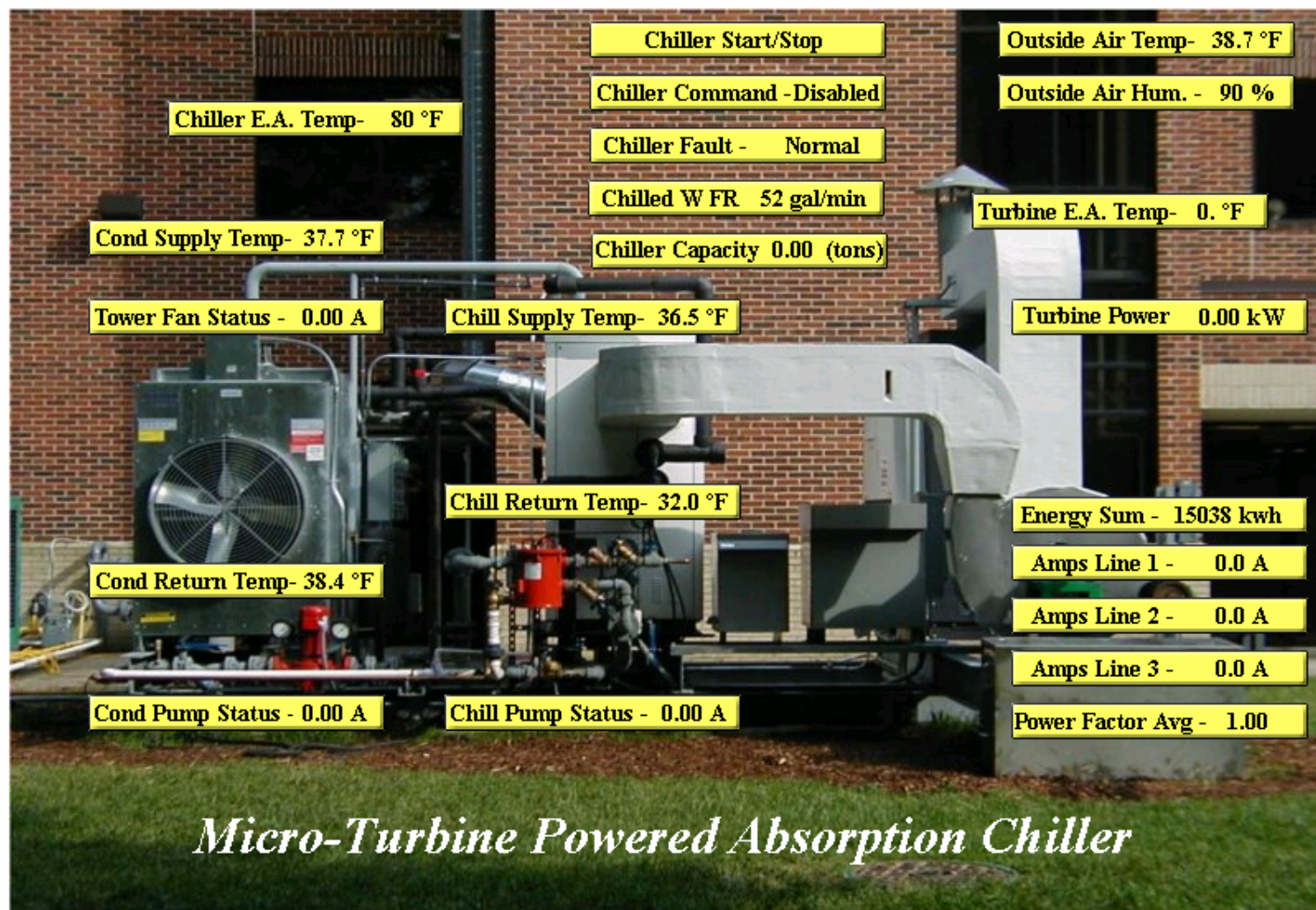


Comparison



Primary Energy Savings

- Compared to existing equipment: 17%
- Compared to new equipment: 5-10%
- Goal for 2002 Cooling Season: as close to 30% as possible!
- Largest Challenge: Parasitic Power



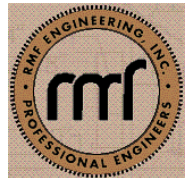
Accomplishments

- ➡ Two Systems Installed and in Operation
- ➡ Education and Information Dissemination
 - ➡ Tours, Conferences
 - ➡ Workshops, 1 Graduate, 1 Undergrad. Course
- ➡ Improved Indoor Air Quality: Reduced Humidity, Other Aspects under Investigation
- ➡ Lowered Electric Power Consumption
- ➡ Primary Energy Savings
- ➡ Current/Future Work
 - ➡ Minimize parasitic loads
 - ➡ Controls Integration
 - ➡ Complete System Performance Analysis

Professional Collaboration

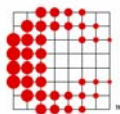
- ➡ ORNL – Sensors (CO₂ and Humidity)
- ➡ PNNL – Whole Building Diagnostician
- ➡ NREL – Liquid Desiccant Components
- ➡ Energy Storage (DOE, Energetics, NRECA, Sandia – Distributed Energy Technology Simulator)
- ➡ Southern Research Institute, EPA, Honeywell – Independent Verification of Micro-Turbine Performance and Emissions
- ➡ ORNL – Integrated System Performance Evaluation

Thanks !



Pacific Northwest
National Laboratory

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U.S. Department of Energy



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